The Consultant Engineer

There was an engineer who had an exceptional gift for fixing all mechanical things. After serving his company loyally for over 30 years, he happily retired. Several years later his company contacted him regarding a seemingly impossible problem they were having with one of their multi-million dollar machines. They had tried everything and everyone else to get the machine fixed, but to no avail. In desperation, they called on the retired engineer who had solved so many of their problems in the past.

The engineer reluctantly took the challenge. He spent a day studying the huge machine. At the end of the day he marked a small X in chalk on a particular component of the machine and proudly stated, 'This is where your problem is!' The part was replaced and the machine worked perfectly again.

The company received a bill for \$50,000 from the engineer for his services. They demanded an itemized accounting of his charges. The engineer responded briefly:

One chalk mark \$1

Knowing where to put it \$49,999

1

Business

- This is lecture #18 out of 36
 - Half way!
- Exam next week
 - Available at review on Monday (Oct. 17)
 - Due Friday, Oct. 21 at 9 am
 - 3 hrs
 - Take home
 - Practice exam posted on Learning Suite
- Quiz

Review/Quiz

- What is vapor pressure?
- What is partial pressure?
- What is the equation for Raoult's law?

3

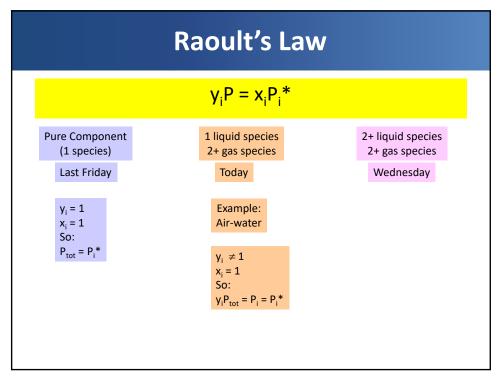
Goals

- Review: Vapor Pressure, Raoult's Law
- Definitions for 1 liquid species, 2+ gas species
 - Dew Point
 - Boiling Point
 - Relative and Absolute Humidity
 - Degrees Superheat
- Examples





Δ



What is Saturation?

• Think of a sponge



- The maximum amount of water absorbed by the sponge is called saturation
- Any extra water after saturation runs off the sponge

Saturated Air

- What happens when air becomes saturated with water vapor?
 - Any additional water starts to condense and form liquid water droplets
 - Point at which the first drop condenses is called the dew point
 - Saturation point (and hence dew point) is a function of temperature (related to vapor pressure)
 - Higher temperature air can hold more water vapor

7

Dew Point Equations

- Dew point is when first drop condenses
 - Saturated Air
 - Occurs when $y_i P_{tot} = P_i^* (x_i = 1)$
 - So air is saturated when $P_i = P_i^*$
 - P_i* is sometimes called P_i^{sat}



- Examples:
 - If you know the temperature, you can calculate $y_{\rm H2O}$ from $P_{\rm i}^*$ and $P_{\rm tot}$, or
 - If you know y_i and P_{tot} , you can calculate P_i^* and then calculate the corresponding temperature (T_{dp})

What if you are not at the dew point?

- How do you quantify the amount of moisture in the air?
 - Mole or mass fraction of H₂O
 - Relative Humidity = $\frac{P_i}{P_i^*} = \frac{P_{H2O}}{P_{H2O}^*}$
 - Like the percent of saturation
 - 50% humidity = 50% of the moisture that the air can hold at that temperature
 - P_{H2O}^* is a function of temperature
 - Saturation is 100% relative humidity
 - Absolute Humidity = $\frac{mass\ of\ vapor}{mass\ of\ dry\ gas}$
 - · Kind of like a weird mass fraction

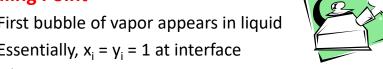
More Definitions

(1 liquid species, 2+ gas species)

- Boiling Point
 - First bubble of vapor appears in liquid
 - Essentially, $x_i = y_i = 1$ at interface
 - $-P_{H2O}^* = P_{tot}$ (same as for 1 gas species)



- Measure of how far you have to cool the gas to condense the first drop of liquid
- Degrees superheat = T T_{dn}



More Definitions

(1 liquid species, 2+ gas species)

- What if P_i > P_i*
 - Supersaturated (relative humidity > 100%)
 - Not at equilibrium
 - Liquid will condense out of the gas phase
 - Conditions will change back to the saturation point where P_i = P_i*

11

Examples

- 1. BYU Heating Plant
- 2. Boiling point in Provo
- 3. Relative humidity in Virginia
 - 1. P_{H2O}
 - 2. y_{H2O}
 - 3. Degrees superheat
- 4. Air conditioner with reheat

Raoult's Law		
	$y_i P_{tot} = x_i P_i^{**}$	$y_i \hat{\boldsymbol{\phi}}_i P = x_i \gamma \boldsymbol{\phi}_i^* P_i^* \exp \left[\sum_{p_i}^{P} \frac{V_i^L}{RT} dP \right]$
Single Component	Liquid = 1 component Gas = 2+ components (n _g)	Liquid = 2+ components (n _L) Gas = 2+ components (n _g)
DF = 2 + 1 - 2 = 1	$DF = 2 + n_{spec} - 2 = n_{spec}$	$DF = 2 + n_{spec} - 2 = n_{spec}$
$y_i = 1.0$ $x_i = 1.0$	$y_i \neq 1.0$ $x_i = 1.0$	$y_i \neq 1.0$ $x_i \neq 1.0$
At vapor-liquid equilibrium, $P_i{}^* = P_{lot}$	At vapor-liquid equilibrium, $y_i P_{lot} = P_i^*$	At vapor-liquid equilibrium, $y_i P_{tot} = x_i P_i^*$
boiling T = condensing T		1st drop of liquid = dew point (saturated vapor)
	boiling when P _{tot} = P _i * (like single component, saturated liquid)	1st bubble of gas = bubble point (saturated li qui d)
	$ \begin{array}{c} \text{Relative humidity} = y_i P_{tot} / P_i^* \text{(or } P_i / P_i^*) \\ \text{- saturated when R.H.} = 100\% \end{array} $	No boiling point defined - temperature changes as lighter component evaporates
P vs T graph valid	Absolute humidity = $\mathbf{m}_{\text{steam}}/\mathbf{m}_{\text{dry gas}}$ = $\left(\frac{P_i M W_i}{R T}\right) / \left(\frac{(P_{tot} - P_i) M W_{dry}}{R T}\right)$ = $P_i M W_i / \left((P_{tot} - P_i) M W_{dry}\right)$	
P vs V graph valid	Degrees superheat = $T \cdot T_{dp}$	

Upcoming HW problems

17.1. Air at 90°C and 1.00 atm (absolute) contains 10.0 mole% water. A continuous stream of this air enters a compressor-condenser, in which the temperature is lowered to 15.6°C and the pressure is raised to 3.00 atm. The air leaving the condenser is then heated isobarically to 100°C. Calculate the fraction of water that is condensed from the air, the relative humidity of the air at 100°C, and the ratio of volumetric flow rates (i.e., m³ outlet air at 100°C per m³ feed air at 90°C).

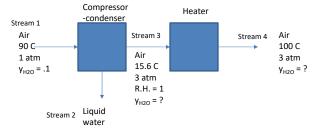
Can ideal gas be used for this calculation?

15

17.1. Air at 90°C and 1.00 atm (absolute) contains 10.0 mole% water. A continuous stream of this air enters a compressor-condenser, in which the temperature is lowered to 15.6°C and the pressure is raised to 3.00 atm. The air leaving the condenser is then heated isobarically to 100°C. Calculate the fraction of water that is condensed from the air, the relative humidity of the air at 100°C, and the ratio of volumetric flow rates (i.e., m³ outlet air at 100°C per m³ feed air at 90°C).

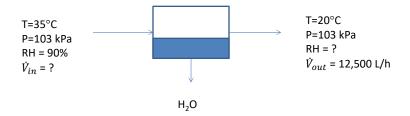
Can ideal gas be used for this calculation?





- 1. Use a basis of 100 moles of moist air entering compressor
- 2. Find $y_{H2O,3}$ from vapor pressure, RH₃, and definition of partial pressure
 3. Note that $y_{H2O,3} = \frac{n_{H2O,3}}{n_{H2O,3} + n_{dry\; air'}}$, so if you know $y_{H2O,3}$ and $n_{dry\; air'}$ you can get $n_{H2O,3}$
- 4. Do the balances on moles of water in and out of compressor
- 5. Use ideal gas law to get ratios of volumetric flow rates

17.2. On a hot summer day the temperature is 35°C, barometric pressure is 103 kPa, and the relative humidity is 90%. An air conditioner draws in outside air, cools it to 20°C, and delivers it at a rate of 12,500 L/h. Calculate the rate of moisture condensation (kg/h) and the volumetric flow rate of the air drawn from the outside.



Strategy:

- 1. Find P* for inlet, then y_{H2O,in}
- 2. Recognize that outlet air is saturated with water (RH = 100%), so find P* for outlet and y_{H2O,out}
- 3. From outlet volumetric flow rate, find $n_{gas,out}$, then $n_{H2O,out}$
- 4. $n_{air,in} = n_{air, out}$, so find $n_{H2O,in}$
- 5. n_{H2o,liq} by difference

- 17.3. A hot-air dryer is used to reduce the moisture content of 1500 kg/min of wet wood pulp from 0.75 kg H₂O/kg dry pulp to 0.15 wt% H₂O. Air is drawn from the atmosphere at 28°C, 760 mm Hg, and 50% relative humidity, sent through a blower-heater, and then fed to the dryer. The air leaves the dryer at 80°C and 10 mm Hg (gauge). A sample of the exit air is drawn into a chamber containing a mirror and cooled slowly, keeping the gauge pressure at 10 mm Hg. A mist is observed to form on the mirror at a temperature of 40.0°C.
- (a) What is the vapor pressure of H₂O at the conditions in the dryer exit?
- (b) Calculate the partial pressure and mole fraction of H₂O in the air leaving the dryer.
- (c) Calculate the mass of water removed from the pulp (kg/min) and the volumetric flow rate of air entering the system (m³/min).

