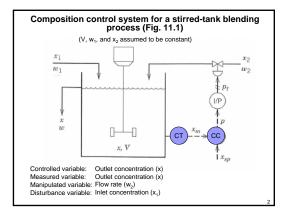
# Dynamic Behavior and Stability of Closed-Loop Control Systems

- We now want to consider the dynamic behavior of processes that are operated using feedback control.
- The combination of the process, the feedback controller, and the instrumentation is referred to as a *feedback control loop* or a *closed-loop system*.

## **Block Diagram Representation**

To illustrate the development of a block diagram, we return to a previous example, the stirred-tank blending process considered in earlier chapters.



#### Process

In section 4.3 the approximate dynamic model of a stirred-tank blending system was developed:

$$\rho V \frac{dx}{dt} = w_1 x_1 + w_2 x_2 - (w_1 + w_2) x$$

$$f = w_1 x_1 + w_2 x_2 - (w_1 + w_2) x$$

$$\left[\frac{\partial f}{\partial x_1}\right]_{ss} = \overline{w}_1$$

$$\left[\frac{\partial f}{\partial w_2}\right]_{ss} = \overline{x}_2 - \overline{x} = 1 - \overline{x} \qquad (x_2 \equiv 1)$$

$$\left[\frac{\partial f}{\partial x}\right]_{ss} = -(\overline{w}_1 + \overline{w}_2)$$

Combining partial fractions and deviation variables,

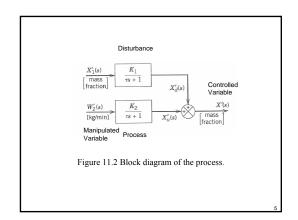
$$\rho V \frac{dx'}{dt} = \overline{w}_1 x_1' + (1 - \overline{x}) w_2' - (\overline{w}) x'$$

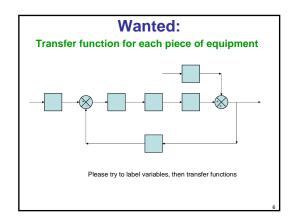
$$\left(\frac{\rho V}{\overline{w}} s + 1\right) X'(s) = \frac{\overline{w}_1}{\overline{w}} X_1'(s) + \frac{(1 - \overline{x})}{\overline{w}} W_2'(s)$$

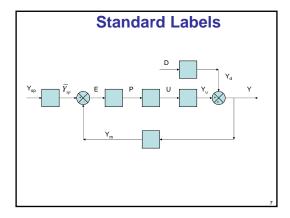
$$X'(s) = \frac{K_1}{\tau s + 1} X_1'(s) + \frac{K_2}{\tau s + 1} W_2'(s)$$

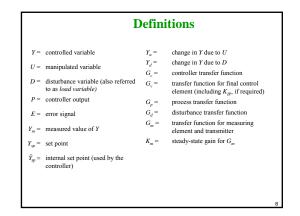
where

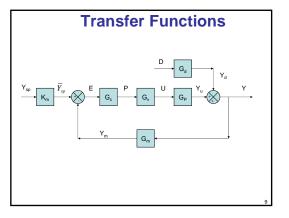
$$\tau = \frac{V\rho}{\overline{w}}, \quad K_1 = \frac{\overline{w}_1}{\overline{w}}, \quad and \quad K_2 = \frac{1-\overline{x}}{\overline{w}}$$

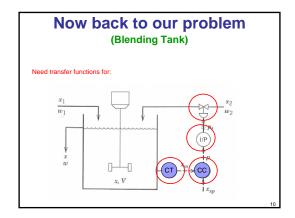


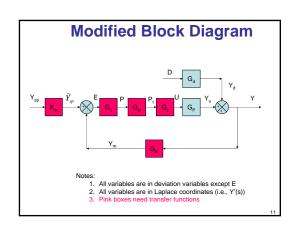












## **Composition Sensor-Transmitter (Analyzer)**

We assume that the dynamic behavior of the composition sensortransmitter can be approximated by a first-order transfer function:



e approximated by a first-order transfer function 
$$\frac{X'_m(s)}{X'(s)} = \frac{K_m}{\tau_m s + 1}$$
 (11-3)

### Controller

Suppose that an electronic proportional plus integral controller is used. From Chapter 8, the controller transfer function is



$$\frac{P'(s)}{E(s)} = K_c \left( 1 + \frac{1}{\tau_I s} \right) \tag{11-}$$

where P'(s) and E(s) are the Laplace transforms of the controller output p'(t) and the error signal e(t). Note that p' and e are electrical signals that have units of mA, while  $K_c$  is dimensionless. The error signal is expressed as

$$e(t) = \tilde{x}'_{sp}(t) - x'_{m}(t)$$
 (11-5)

or after taking Laplace transforms,

$$E(s) = \tilde{X}'_{sp}(s) - X'_{m}(s)$$
 (11-6)

The symbol  $\vec{x}_{sp}'(t)$  denotes the *internal set-point* composition expressed as an equivalent electrical current signal. This signal is used internally by the controller  $\vec{x}_{sp}'(t)$  is related to the actual composition set point  $\vec{x}_{sp}'(t)$  by the composition sensor-transmitter gain  $K_{mi}$ :

$$\tilde{x}_{sp}'\left(t\right) = K_{m}x_{sp}'\left(t\right) \tag{11-7}$$

Thus



$$\frac{\tilde{X}'_{sp}(s)}{X'_{sp}(s)} = K_m \tag{11-8}$$

## Current-to-Pressure (I/P) Transducer

Because transducers are usually designed to have linear characteristics and negligible (fast) dynamics, we assume that the transducer transfer function merely consists of a steady-state gain  $K_m$ :



$$\frac{P_t'(s)}{P'(s)} = K_{IP} \tag{11-9}$$

### **Control Valve**

As discussed in Section 9.2, control valves are usually designed so that the flow rate through the valve is a nearly linear function of the signal to the valve actuator. Therefore, a first-order transfer function usually provides an adequate model for operation of an installed valve in the vicinity of a nominal steady state. Thus, we assume that the control valve can be modeled as



$$\frac{W_2'(s)}{P_t'(s)} = \frac{K_v}{\tau_v s + 1}$$
 (11-10)

