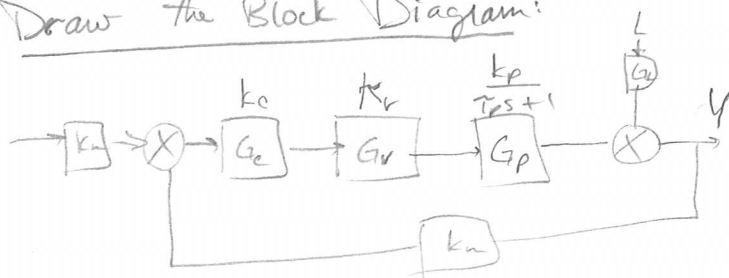


Closed Loop Systems① PROPORTIONAL CONTROL
Simple Case

- 1 - Proportional Control
- 2 - Neglect valve dynamics
- 3 - Neglect measurement dynamics
- 4 - First order process
- 5 - pneumatic controller

A. Draw the Block Diagram:

B. What is Y/Y_{sp} ? (shortcut method)

$$\frac{Y}{Y_{sp}} = \frac{k_m k_c k_v \frac{k_p}{\tau_p s + 1}}{1 + k_m k_c k_v \frac{k_p}{\tau_p s + 1}}$$

rearrange (multiply top & bottom by $\tau_p s + 1$)

$$\frac{Y}{Y_{sp}} = \frac{k_m k_c k_v k_p}{\tau_p s + 1 + k_m k_c k_v k_p}$$

rearrange (divide by $(1 + k_m k_c k_v k_p)$ to get $\tau' s + 1$ form)

$$\frac{Y}{Y_{sp}} = \frac{\frac{k_m k_c k_v k_p}{1 + k_m k_c k_v k_p}}{\frac{\tau_p}{1 + k_m k_c k_v k_p} s + 1} = \frac{\frac{K_{OL}}{1 + K_{OL}}}{\frac{\tau_p}{1 + K_{OL}} s + 1} = \frac{K'}{\tau' s + 1}$$

redefine terms

$$K_{OL} = k_m k_c k_v k_p$$

C. How does the magnitude of τ' compare to τ_p ? Why?ans: if K_{OL} is finite, $\tau' < \tau_p$ How is τ' affected by k_c ?ans. - as $k_c \uparrow$, $\tau' \downarrow$ (faster response)

D. What is the time domain solution to a step change in y_{sp} ?

$$Y(s) = \left(\frac{k'}{\tau's + 1} \right) \left(\frac{M}{s} \right)$$

$$y(t) = k'M (1 - e^{-t/\tau'})$$

E. What is the steady-state value of y_{sp} ? (M)

What is the steady-state value of y ? $k'M = \frac{k_{OL}}{1+k_{OL}} M$

What is offset?

$$\text{offset} = y_{sp}(\infty) - y(\infty) = M - k'M = M(1 - k')$$

$$= M \left(1 - \frac{k_{OL}}{1+k_{OL}} \right) = M \left(\frac{1+k_{OL} - k_{OL}}{1+k_{OL}} \right)$$

$$= M \left(\frac{1}{1+k_{OL}} \right) \quad \text{see Eq. 11-43}$$

F. So for proportional control, how does k_c affect offset?

As $k_c \uparrow$, offset $\downarrow$

G. Summary

As $k_c \uparrow$, offset $\downarrow$, $\tau' \uparrow$

see Figure 11.18

H. Something happens for $\frac{C}{L}$ (responses to step changes in Load)

② PI Control

$$G_c = k_c \left(1 + \frac{1}{T_I s} \right)$$

A. What is y/y_{sp} ?

$$\frac{y}{y_{sp}} = \frac{k_m k_c \left(1 + \frac{1}{T_I s} \right) k_v k_p / (T_p s + 1)}{1 + k_m k_c \left(1 + \frac{1}{T_I s} \right) k_v \frac{k_D}{T_p s + 1}}$$

B. Reduce (multiply by $T_p s + 1$)

$$\frac{y}{y_{sp}} = \frac{k_m k_c k_v k_p \left(1 + \frac{1}{T_I s} \right)}{T_p s + 1 + k_m k_c k_v k_p \left(1 + \frac{1}{T_I s} \right)}$$

C. Multiply top & bottom by $T_I s$

$$\begin{aligned} \frac{y}{y_{sp}} &= \frac{k_m k_c k_v k_p (T_I s + 1)}{T_p T_I s^2 + T_I s + k_m k_c k_v k_p (T_I s + 1)} \\ &= \frac{k_m k_c k_v k_p (T_I s + 1)}{T_p T_I s^2 + T_I (1 + k_m k_c k_v k_p) s + k_m k_c k_v k_p} \end{aligned}$$

D. Place in standard 2nd order form (divide by $k_m k_c k_v k_p$) needs to be 1

$$\frac{y}{y_{sp}} = \frac{T_I s + 1}{\underbrace{\frac{T_p T_I}{k_m k_c k_v k_p}}_{(\tau')^2} s^2 + \underbrace{\frac{T_I (1 + k_m k_c k_v k_p)}{k_m k_c k_v k_p}}_{2\tau' \zeta} s + 1}$$

$$\tau' = \sqrt{\frac{T_p T_I}{k_m k_c k_v k_p}}$$

$$\zeta = \frac{T_I (1 + k_m k_c k_v k_p)}{k_m k_c k_v k_p} \cdot \frac{1}{2 \sqrt{\frac{T_p T_I}{k_m k_c k_v k_p}}}$$

$$= \frac{(1 + k_m k_c k_v k_p)}{2 \sqrt{k_m k_c k_v k_p}} \sqrt{\frac{T_I}{T_p}}$$

E. Stability?

ans. since ζ is positive, zeroes are negative, so stable roots!
(if $\zeta < 1$, underdamped)

② PI control (cont.)

F. So what did the addition of PI control do to the system?
→ 2nd order

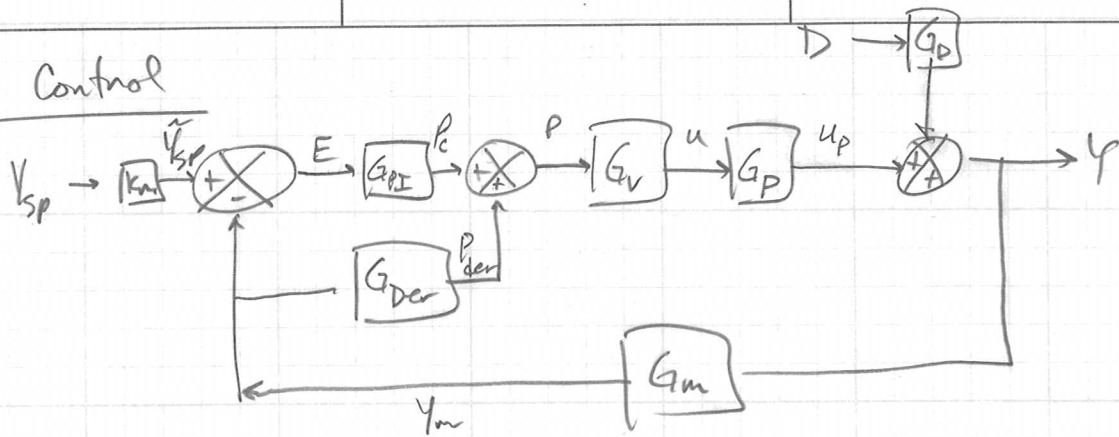
G. What is the steady-state value of Y in response to a unit step change in Y_p ?

$$Y(s) = \frac{(\tau_I s + 1)}{\tau'^2 s^2 + 2\tau' s + 1} \frac{M}{s}$$

$$\lim_{s \rightarrow 0} s Y(s) = M$$

(no offset!)

PID Control



$$Y = G_d D + P G_v G_p$$

$$P = P_c + P_{der}$$

$$P_c = G_{PI} E$$

$$P_{der} = Y_m G_{der}$$

$$E = \tilde{Y}_{sp} - Y_m = K_m Y_{sp} - G_m Y$$

so

$$P = G_{PI} K_m Y_{sp} - G_{PI} G_m Y + \underbrace{Y_m G_{der}}_{G_m Y}$$

$$Y = G_d D + G_v G_p (G_{PI} K_m Y_{sp} - G_{PI} G_m Y + G_m G_{der} Y)$$

$$Y \left(1 + G_v G_p (G_{PI} G_m - G_m G_{der}) \right) = G_d D + G_v G_p G_{PI} K_m Y_{sp}$$

$$\frac{Y}{Y_{sp}} = \frac{K_m G_v G_p G_{PI}}{1 + G_v G_p G_m (G_{PI} - G_{der})}$$

$$G_v = k_v$$

$$G_p = 1^{st} \text{ order} = \frac{k_p}{\tau_p s + 1}$$

$$G_{PI} = K_c \left(1 + \frac{1}{\tau_I s} \right)$$

$$G_{der} = -K_c \tau_{der} s$$

$$G_m = K_m$$

$$\frac{Y}{Y_{sp}} = \frac{K_m K_c \left(1 + \frac{1}{\tau_I s} \right) k_v \frac{K_p}{\tau_p s + 1}}{1 + K_m K_c \left(1 + \frac{1}{\tau_I s} \right) k_v \left(\frac{K_p}{\tau_p s + 1} \right) + K_c \tau_{der} s K_m k_v \left(\frac{K_p}{\tau_p s + 1} \right)}$$

next: (a) multiply top : bottom by $\tau_p s + 1$
(b) multiply top : bottom by $\tau_I s$

(a) multiply by $\tau_p s + 1$

$$\frac{Y}{Y_{sp}} = \frac{k_m k_c k_v k_p \left(1 + \frac{1}{\tau_I s}\right)}{(\tau_p s + 1) + k_m k_c k_v k_p \left(1 + \frac{1}{\tau_I s}\right) + k_c k_m k_v k_p \tau_{Der} s}$$

(b) multiply by $\tau_I s$

$$\frac{Y}{Y_{sp}} = \frac{k_m k_c k_v k_p (\tau_I s + 1)}{(\tau_p s + 1) \tau_I s + k_m k_c k_v k_p (\tau_I s + 1) + k_c k_m k_v k_p \tau_{Der} s (\tau_I s)}$$

now group s^2 terms, s terms, etc.

$$\frac{Y}{Y_{sp}} = \frac{k_m k_c k_v k_p (\tau_I s + 1)}{\tau_p \tau_I s^2 + \tau_I s + k_m k_c k_v k_p \tau_I s + k_m k_c k_v k_p + k_c k_m k_v k_p \tau_{Der} \tau_I s^2}$$

also divide by $k_m k_c k_v k_p = K_{OL}$

$$\frac{Y}{Y_{sp}} = \frac{\tau_I s + 1}{\left(\tau_{Der} \tau_I + \frac{\tau_p \tau_I}{K_{OL}}\right) s^2 + \tau_I \left(1 + \frac{1}{K_{OL}}\right) s + 1}$$

$$\frac{Y}{Y_{sp}} = \frac{\tau_I s + 1}{\left(\tau_{Der} \tau_I + \frac{\tau_p \tau_I}{K_{OL}}\right) s^2 + \tau_I \left(1 + \frac{1}{K_{OL}}\right) s + 1}$$

2nd order $\Rightarrow$ get τ^2, ζ, k