

Chemical Engineering 374

Fluid Mechanics

Compressible Flow



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Spiritual Thought

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Moroni 10:5

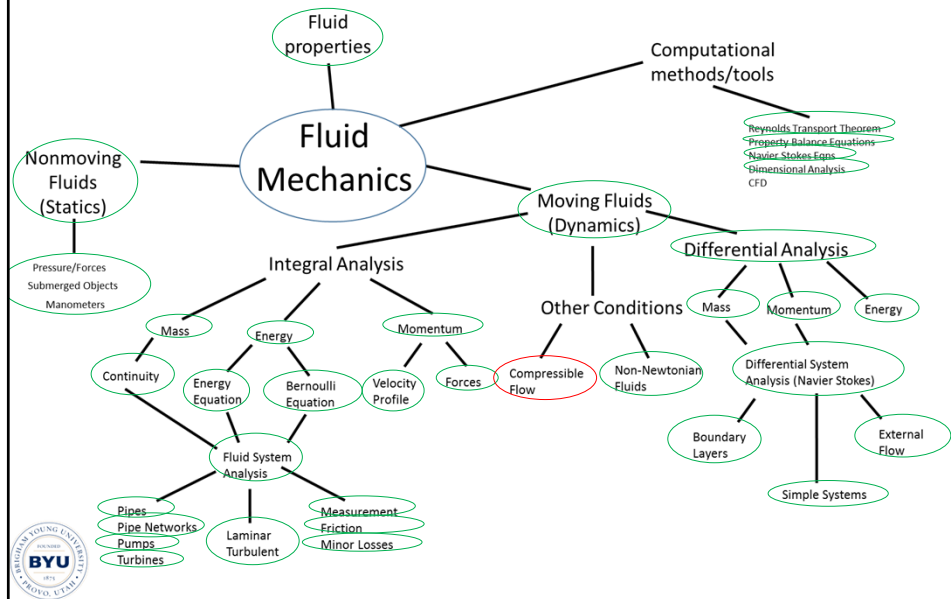
And by the power of the Holy Ghost
ye may know the truth of all things.



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Fluids Roadmap

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Key Points

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- Characteristics
 - Converts u to kinetic energy
 - T drops, p drops
 - v greater than in non-compressible flow
 - Ma
- Equations
- Nozzles
- Choked Flow



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Compressible Flows

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- Previously, ρ was constant
- Now, ρ varies
 - High speed flow, large ΔP , ΔT , Δv , etc.
 - Internal energy is converted to kinetic energy
 - T Drops; ρ drops
 - $v > v$ of Bernoulli Eqn. where ρ is constant
 - Mach number, $Ma = v/c$; c is **local** value, $[c(T)]$
 - Compressible for $Ma > 0.3$, $\rho/\rho \sim 0.95$,

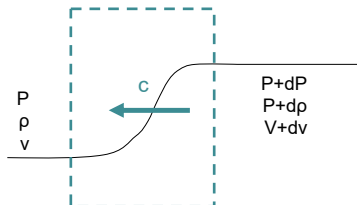


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Sound Speed

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- Pressure Pulse Analysis



- Mass Balance
- Momentum Balance
- Expression for c
- Error



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- Mass Balance

$$\dot{m}_{in} = \dot{m}_{out} \Rightarrow \rho A v = (\rho + \Delta \rho) A (v + \Delta v)$$

$$\rho v = \rho + \rho dv + v dp + d\rho v$$

$$\rho dv = -v dp$$

- Momentum Balance $\rightarrow$ SS, only Pressure forces

$$AP - A(P + dP) = \dot{m}(v + dv) - \dot{m}v \Rightarrow (\dot{m} = \rho A v)$$

$$\dot{m} = \rho A v, A \text{ cancels}$$

$$-dP = \rho v dv \Rightarrow \boxed{\rho dv = -\frac{dP}{v}}$$

$$\frac{dP}{d\rho} = v^2 = c^2 \rightarrow c = \sqrt{\frac{dP}{d\rho}}$$



$$P = \frac{\rho RT}{M}; \frac{dP}{d\rho} \rightarrow \left(\frac{\partial P}{\partial \rho}\right)_T \text{ or } \left(\frac{\partial P}{\partial \rho}\right)_S$$

$$\boxed{c = \sqrt{\left(\frac{\partial P}{\partial \rho}\right)_S} = \sqrt{\frac{kRT}{M}}}$$

- Sound waves compress

- No heat transfer

- Isentropic

- NOT isothermal

$$c = \sqrt{\left(\frac{\partial P}{\partial \rho}\right)_S} = \sqrt{\frac{kRT}{M}}$$

$$k = \frac{C_p}{C_v}$$

$$= 1.4 \text{ for air}$$



Common Values of c

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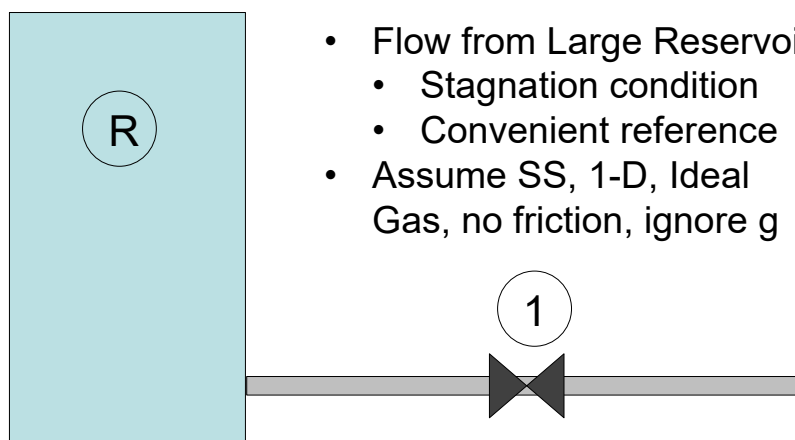
	c (m/s)	c (mph)
Air @ 298 K	346	774
Air @ 2000 K	896	2000
H₂O @ 288 K	1490	3333
Steel @ 188 K	5060	11318



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Compressible Flow

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- Flow from Large Reservoir
 - Stagnation condition
 - Convenient reference
- Assume SS, 1-D, Ideal Gas, no friction, ignore g



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- Energy Balance

- $\dot{Q} + \dot{W}_s = \dot{m} \left(\frac{p}{\rho} + u + \frac{v^2}{2} + gz \right)_R - \dot{m} \left(\frac{p}{\rho} + u + \frac{v^2}{2} + gz \right)_1$

- $v_1 = 2(h_r - h_1) = 2C_p(T_r - T_1)$

- Given: $C_p = C_v + \frac{R}{M} \rightarrow k = C_p/C_v \rightarrow C_v = C_p/k$

- $C_p = \frac{Rk}{M(k-1)}$

- $c^2 = \frac{kRT}{M}$

- $\frac{v_1^2}{c_1^2} = Ma^2 = \frac{2}{k-1} \left(\frac{T_r}{T_1} - 1 \right)$

- $\frac{T_r}{T_1} = \frac{Ma^2(k-1)}{2} + 1$



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- Ideal Gases: (thermo, next semester...)

- $\frac{P_r}{P_1} = \left(\frac{T_r}{T_1} \right)^{k/(k-1)}$; $\frac{\rho_r}{\rho_1} = \left(\frac{T_r}{T_1} \right)^{1/(k-1)}$

- Thus,

- $\frac{P_r}{P_1} = \left(Ma^2 \frac{k-1}{2} + 1 \right)^{k/(k-1)}$

- $\frac{\rho_r}{\rho_1} = \left(Ma^2 \frac{k-1}{2} + 1 \right)^{1/(k-1)}$

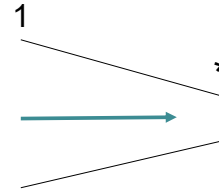


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Nozzles

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- Mass flow = constant
- As A decreases, v increases
- Eventually $v = c$ at (*)
 - $A_1 \rho_1 v_1 = A^* \rho^* v^*$
 - $A_1 / A^* = \rho^* v^* / \rho_1 v_1$



$$\frac{A_1}{A^*} = \frac{1}{Ma_1} \left[\frac{Ma_1^2 \frac{(k-1)}{2} + 1}{\frac{(k-1)}{2} + 1} \right]^{\frac{(k+1)}{2(k-1)}}$$

Handwritten notes: $M = \frac{v}{c}$, $\frac{A_1}{A^*}$, m_1 , plot

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- For $M < 1$, as velocity increases, A decreases
- For $M > 1$, as velocity increases, A increases!
 - Because ρ is decreasing faster than A is increasing $\rightarrow v$ increase to keep $\dot{m} = \rho A v$ constant



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Choked Flow

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- Flow Changes “communicated” at speed of sound
- i.e. pressure waves – travel at c
- If flow speed increases to c , then communication wave can’t get back to reservoir
 - Shouting in wind
 - “Communication” doesn’t occur
- Flow doesn’t increase not matter how low P_2 becomes
 - Choked flow
 - Gas Control valves, orifices, vacuum systems, bike tires
 - Choke point is minimum area in the nozzle/valve
- Flow is constant until ΔP decreases below Critical pressure



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